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1 A LIGHTING FIXTURE

2
3 CROSS-REFERENCE TO RELATED APPLICATION

4 This application claims the benefit, under 35 U.S.C. §119(e), of U.S.
5 Provisional Application No. 60/201,489; filed May 3, 2000.

6
7 FEDERALLY-SPONSORED RESEARCH OR DEVELOPMENT

8 Not Applicable

9
10 BACKGROUND OF THE INVENTION

11 The present invention relates to a lighting fixture for projecting a beam
12 of light and for use for spot lighting in connection with theater stages, cinema
13 and television studios and the like, the fixture comprising:

14 a light source arranged at one end of a housing having a light beam exit
15 aperture at the opposite end thereof, the light source and aperture being
16 arranged generally concentric with a longitudinal or optical axis of the
17 lighting fixture,

18 light beam influencing means at least comprising one or more,
19 preferably four, beam-shaping blades and preferably also comprising other
20 light influencing means such as one or more lenses and/or an iris and/or a
21 pattern or gobo, for influencing a light beam emitted by the light source and
22 being arranged along the path of the light beam along said longitudinal axis
23 through the housing from the light source to the aperture, and

24 adjustment means for adjusting the position of at least said one or more
25 beam-shaping blades and preferably of all said influencing means relative to
26 said longitudinal axis.

27 *The purpose of a lighting fixture as defined above is to produce a well-
28 defined light beam or light cone with a geometry, angle of conicity and focal
29 point that may be altered manually or by remote control.

1 A lighting fixture will normally comprise a light source, a reflector, a
2 beam-shaping gate with beam-shaping blades, a pattern or gobo, an iris, a
3 focusing lens, a zoom lens and a color filter as well as a suspension structure
4 allowing the lighting fixture to be pivoted vertically and horizontally.

5 The visible part of the light emitted by the light source is collected by
6 the reflector and is sent towards the iris, the gobo and the beam-shaping gate
7 as a parallel light beam. The infrared part of the radiation from the light
8 source passes through the dichroic coating of the reflector and impinges on
9 the inner surface of the housing surrounding the light source, the heat being
10 transported to the outer surface of the housing having cooling ribs for emitting
11 the heat to the surrounding atmosphere.

12 It is often necessary to be able to determine the geometry of the light
13 beam, and this is achieved by means of the zoom lens varying the angle of
14 conicity of the light cone and by shaping or cutting off the periphery of the
15 light beam by means of the beam-shaping gate with beam-shaping blades so
16 as to obtain geometrical figures such as squares, triangles, trapezoids etc. The
17 lenses project the light out through the aperture of the housing opposite the
18 light source and through the color filter at the front end of the lighting fixture.
19 It is important that the different elements influencing the shape and other
20 characteristics of the light beam function as precisely as possible even when
21 being influenced by the heat radiated from the light source and not removed
22 by means of the dichroic reflector. This entails that the location and the
23 configuration of the adjustment means for the beam-shaping blades, the gobo
24 and iris are such that any bending caused by the heat influence from the light
25 beam be kept at a minimum.

26 Lighting fixtures of this type are often arranged in places where it is
27 difficult to access them manually and it is therefore of great importance that
28 the adjustment means for adjusting the above-mentioned beam influencing
29 means be as easily accessed and as flexible as possible when manual operation

1 comprises remote sensing means for sensing the angular position of said
2 element relative to said longitudinal axis.

3 So as to obtain the greatest flexibility of adjustment and the greatest
4 range of adjustment, the adjustment means for each of the one or more beam-
5 shaping blades comprises radial adjustment means for adjusting the position
6 of the blade radially relative to said axis, and circumferential adjustment
7 means for adjusting the position of said blade circumferentially around said
8 axis.

9 A particularly simple and efficient as well as accurate embodiment of
10 the light fixture according to the invention is provided by the adjustment
11 means for each of the one or more beam-shaping blades comprising two
12 adjacent co-centrical annular bodies or rings each connected to one point of
13 the blade such that relative rotation of the two rings alters the radial position
14 of the blade.

15 In the currently preferred embodiment, the rings comprise guiding
16 tracks recessed into the lateral surface of each ring facing the other ring, and
17 each blade comprises a body extending generally transversely to said axis and
18 two arms extending generally parallel to said axis, the arms each being
19 provided with sliding connecting means for connecting the respective arm to
20 each of the rings and being adapted for being slidingly received in a guiding
21 track in each of said rings.

22

23 BRIEF DESCRIPTION OF THE DRAWINGS

24 In the following description, preferred embodiments of a lighting
25 fixture according to the invention will be described in detail, solely by way of
26 example, with reference to the accompanying drawings, where:

27 Fig. 1 is an isometric elevational view of a lighting fixture according to
28 the invention for manual operation;

29 Fig. 2 is a partially cut-away view of the lighting fixture in Fig. 1

1 illustrating the internal configuration of the lighting fixture;

2 Fig. 3 is a schematic cross-sectional view of the lighting fixture of Figs
3 1 and 2, the cross-section being taken along a vertical plane containing the
4 longitudinal or optical axis of the lighting fixture;

5 Fig. 4 is an enlarged scale view of the left-hand part of Fig. 3;

6 Fig. 5 is an isometric elevational view of the bottom half of the frame
7 of the lighting fixture of Figs. 1 and 2;

8 Fig. 6 is an exploded view of the beam-shaping blades and adjustment
9 rings of the fixture in Figs. 1 and 2;

10 Fig. 7 is an axial end view of the blades and rings shown in Fig. 6 in
11 nested assembled condition;

12 Figs. 8 and 9 are schematic axial end views corresponding to Fig. 7
13 illustrating the adjustment of the beam-shaping blades of Figs. 6-7;

14 Fig. 10 is an illustration of the constructive principles of the guiding
15 tracks in the adjustment rings for the beam-shaping blades;

16 Fig. 11 schematically illustrates an alternative embodiment of the
17 beam-shaping blades and the adjustment mechanisms therefor;

18 Fig. 12 shows an isometric partly exploded view in larger scale of the
19 position adjustment mechanism for the lenses shown in Fig. 2; and

20 Fig. 13 shows an enlarged view of a detail of the construction shown in
21 Fig. 12.

22

23 DETAILED DESCRIPTION OF THE INVENTION

24 Referring now to Figs 1-5, a lighting fixture 1 according to the
25 invention is suspended in a suspension fitting 2 having an aperture 3 for
26 fixing the fitting 2 pivotably to a support structure (not shown) in a theater, a
27 television studio or the like. The fitting 2 is pivotably attached to the body of
28 the lighting fixture 1 at 4, the attachment point being adjustable by sliding the
29 pivot attachment point 4 in a slit 5 in a frame 6 so as to compensate for change

1 of balance because of insertion or removal of different elements in the
2 lighting fixture 1.

3 The lighting fixture 1 may thus be manually pivoted around two
4 mutually substantially orthogonal axes allowing the direction of a light beam
5 emitted by the fixture to be any desired direction.

6 If it is desired to be able to remotely control the direction of the beam,
7 the pivoting action may be achieved by means of remotely controlled
8 electrical motors in many different ways that will be obvious to those skilled
9 in the art.

10 The frame 6 is generally U-shaped having two arms supporting the
11 body of the lighting fixture 1 between said arms. A series of toothed rims 7-18
12 are arranged for rotation around a longitudinal or optical axis 19 (see Fig. 3).
13 The teeth of the toothed rims are configured such that the teeth of a pinion of
14 a drive unit may engage and mesh therewith if the light beam influencing
15 facilities of the lighting fixture operated by rotation of the bodies 7-18 are to
16 be motorized for remote control.

17 In the manually operated embodiment shown in Figs 1-4, the teeth of
18 the toothed rims serve as a roughening element of the surface of the rim of
19 each of the annular bodies 7-18 such that good frictional engagement between
20 the fingers of a hand and the toothed rims or annular bodies 7-18 may be
21 achieved for rotating the annular bodies 7-18 manually.

22 Such roughening of the rim surface may be achieved in many other
23 ways such as scoring of the surface or coating with rubber or provision of
24 small projections etc.

25 In such case and if motorization of the rotation of the bodies 7-18 is
26 desired, then a frictional surface engagement of for instance the surface of a
27 rubber coated drive wheel driven by an electrical motor with the roughened
28 rim surface may be provided for instead of the meshing of the teeth of a
29 pinion with teeth of the rim of the annular body.

1 A light source or lamp 20 emits a light beam composed of individual
2 light beams such as illustrated at 20a, the visual portion thereof being
3 reflected by a dichroic reflector 21 through a focusing lens 22 and a zoom lens
4 23 and out of the lighting fixture through an aperture 24 in the housing 25 of
5 the fixture 1, the light beam 20a traveling through a color filter (not shown)
6 arranged in four color filter holders 26 that may be pivoted around pivots 27
7 so as to allow a color filter to be inserted and removed in the holders 26 in any
8 of four directions determined by the four holders 26. Hereby the color filter
9 may be inserted and removed from the best angle for manual access for a
10 given orientation of the housing 25. The entire light beam projected by the
11 lighting fixture is of course composed of a plurality of light beams analogous
12 to individual light beam 20a.

13 The infra red portion of the light beam 20a is transmitted through the
14 dichroic reflector 21 to cooling ribs 22 in a manner well known in the art so as
15 to reduce the heat distortion of light beam influencing elements, as described
16 below, that are arranged along the path of the light beam from the light source
17 20 to the exit aperture 24.

18 These light beam influencing elements comprise an iris 28 connected
19 to the annular body 7, a pattern or gobo 29 connected to the annular body 8,
20 four beam-shaping blades 30, 31, 32 and 33 connected to the pairs of annular
21 bodies, 9 -10, 11-12, 13 -14 and 15 -16, respectively, the focusing lens 22
22 connected to the annular body 17, and the zoom lens 23 connected to the
23 annular body 18.

24 The annular bodies or rings 7-18 are connected in different manners to
25 the respective light beam influencing elements 22, 23 and 28-33 so that the
26 position of these elements may be altered relative to the axis 19, and thus the
27 light beam, by rotating the rings around said axis. The individual connections
28 between the individual rings and the respective elements will be described
29 more in detail below.

1 The feature of being able to alter the position of the light beam
2 influencing elements, and particularly of the light beam shaping blades 30-33,
3 by means of rotating the corresponding rings allows the position alteration to
4 be carried out manually from a convenient angle of approach for a given
5 orientation of the housing 25. As the rim surface of each of the rings 7-18 may
6 be engaged manually at most of the extent of the circumference thereof, the
7 manual adjustment of the position of a respective light beam influencing
8 element may be performed from the most convenient angle of approach to the
9 housing 25. Furthermore, the manual adjustment may be carried out with one
10 hand which is important, as the fixture is often located such that access with
11 both hands is difficult and perhaps impossible.

12 Hereby the lighting fixture according to the invention does not have the
13 disadvantages of known lighting fixtures where the adjustment means for
14 adjusting the position of a light beam shaping blade may be very
15 inconveniently located relative to the position of the person operating the
16 lighting fixture so that the person for instance has to reach around the lighting
17 fixture housing to access the adjustment means thereby risking being burned
18 on the hot housing surface and rendering rapid and precise position
19 adjustment difficult and perhaps impossible.

20 This advantage can also be obtained by rotational means other than
21 rings with a rim surface for being engaged manually or mechanically.
22 Elements having a plurality of radially extending spokes spaced
23 circumferentially for being engaged at the ends thereof by fingers of a hand or
24 a motorized driving means may also be used. A circumferentially disposed
25 endless belt arranged for substantially circular movement around the
26 longitudinal axis may also be utilized instead of the illustrated rings. All
27 means allowing access along a major part of the circumference of the housing
28 and rotational frictional engagement by fingers or a motorized drive unit may

1 be used to allow such convenient access to the adjustment means for altering
2 the position of the beam influencing elements.

3 The feature of altering the position of the light influencing elements by
4 rotational means also entails simple and reliable establishment of a certain
5 adjustment setting of a respective influencing means such that pre-
6 programmed settings may be set up for certain lighting requirements knowing
7 that it will be simple, quick and reliable to achieve such settings either
8 manually or remotely under difficult conditions, for instance during the course
9 of a theater show where adjustments in the dark are necessary.

10 A further advantage is obtained by the shown structure according to the
11 invention in that the construction is such that no light is emitted from the
12 interior of the fixture except through the aperture 24, and all adjustments of
13 the light beam influencing elements may be carried out without creating a
14 light emission slit or aperture. Hereby, the disadvantage of all known lighting
15 fixtures that light "leaks" therefrom is eliminated which is of great value,
16 particularly for theater use.

17 Referring again to Figs. 1- 5, the frame 6 is constituted by two identical
18 halves 6a and 6b abutting each other at 6c. The rings or annular bodies 7-18
19 are rotatably and slidingly supported in annular grooves 34 in annular support
20 rings 35 by means of annular projections or ridges 36 slidingly received in the
21 annular grooves 34. The support rings 35 are each constituted by half a ring
22 fixedly attached to or made in one piece with one half of the frame 6, for
23 instance 6a (see Fig. 1). In other words each of the frame halves 6a and 6b is
24 fixedly attached to or integral with a series of half rings 35 as shown in Fig.
25 5, where the bottom half 6b of the frame 6 is shown with the corresponding
26 half rings 35.

27 When assembling the lighting fixture 1, the adjustment rings 7-18 with
28 corresponding beam influencing elements 22, 23 and 28-33 are arranged in
29 the bottom half 6b of the frame with corresponding half rings 35 such that the

ridge 36 of each adjustment ring is received in the corresponding groove 34 of the respective half ring 35 of the bottom frame half 6b. Thereafter the top half 6a of the frame 6 with corresponding half rings 35 is placed abutting the bottom half 6b at 6c such that the ridge 36 of each adjustment ring is received in the corresponding groove 34 of the respective half ring 35 of the top frame half 6a. The adjustment rings 7-18 will thus be slidingly and rotationally supported along the entire circumference thereof by the corresponding rings 35.

Each of the adjustment rings or annular bodies 7-18 may then be rotated manually or by means of suitable mechanical means by applying a tangential force to the rim of the respective adjustment ring whereby the ridge 36 thereof slides in the respective annular groove 34 of the respective support ring 35. The material of the ridges 36 and the grooves 34 are chosen such that frictional sliding resistance is kept at a minimum. The support rings 35 may be made of cast aluminum, and the adjustment rings may be made of glass-fiber reinforced plastic. The ridges 36 are made of a low frictional material such as PTFE (marketed, for example, under the trademark "TEFLON"), a ring of said material being embedded in the lateral surface of the corresponding adjustment ring. Hereby the frictional sliding resistance between the low friction material and the cast aluminum will be low, and the adjustment rings may consequently be rotated by applying a relatively small tangential force to the rim thereof.

Each of the adjustment ring pairs 9/10, 11/12, 13/14 and 15/16 carries a respective light beam shaping blade 33, 32, 31 and 30, respectively, by means of pairs of arms 33a,b, 32a,b, 31a,b and 30a,b, respectively, held by the adjustment ring pairs in a manner described more in detail below. So that the two rings of each ring pair can rotate relative to one another, a low friction material ring 37 is arranged between each pair of adjustment rings as illustrated in Figs. 4 and 6.

1 Referring now to Figs. 4 and 6-9, the arrangement of the four light
2 beam shaping blades 30-33 will now be explained more in detail.

3 The blades 30-33 are nested as illustrated in Figs. 4, 6 and 7, each
4 blade 30-33 being carried by a pair of opposed arms, 30a-33a and 30b-33b,
5 respectively. It is important that the blades 30-33 are located as axially close
6 to each other as possible so as to achieve a sharp cut-off boundary of the light
7 beam all around the circumference thereof which only can be achieved if the
8 blades are arranged such that there is no substantial distance between them in
9 the axial direction of the housing. This is particularly well illustrated in Figs. 3
10 and 4 where it is evident that the spacing of the blades in the direction of the
11 axis 19 is slight.

12 The arrangement shown also has the advantage that the axial distance
13 between the beam-shaping blades 30-33 and the iris 28 as well as the gobo or
14 pattern 29 is small so that a good sharpness or quality of the influence of the
15 blades, the iris and the gobo on the light beam may be obtained
16 simultaneously because of the small axial distance covered by all said
17 elements.

18 The blades 30-33 are shaped as shown in Figs. 6-8 having a generally
19 elliptical planar body 38 with an aperture 39 having a periphery comprising a
20 curved portion 40 and linear portions 41, 42 and 43, said periphery serving as
21 the beam cut-off edge of the blade body 38. This is illustrated in Fig. 7 where
22 the peripheries of the apertures 39 of the four bodies 38 of the blades 30-33
23 define the periphery of the beam shaping aperture 44. A multitude of
24 different shapes of the aperture 44 may be achieved by a combination of a
25 rotation of the different blades 30-33 around the axis 19 with a displacement
26 of said blades 30-33 radially relative to said axis 19.

27 The radial displacement of the individual blades 30-33 is illustrated in
28 Figs. 8-9 where the periphery portion 42 of blade 33 is shown in Fig. 8 at the
29 maximum radial distance from the axis 19 and in Fig. 9 at the minimum radial

1 distance from said axis 19. The rotational displacement is achieved by
2 rotating the ring pair 9/10 carrying the blade 33 around the axis 19.
3 Combinations of the radial and the rotational displacement of each blade
4 allow the creation of a great variety of peripheral shapes for the aperture 44.

5 The elliptical shape of the 39 has been chosen to give a relatively stiff
6 blade as well as a continuous and smooth outer perimeter of the body. Hereby
7 the bodies of the blades will not interfere with one another when they are
8 displaced relative to one another even though the axial spacing of the bodies
9 is small. So as to avoid such mutual interference between the bodies as well as
10 between the pairs of arms 30a,b-33a,b it is advantageous that the radial
11 displacement of the bodies take place in such a manner that practically no
12 flexing of the arms takes place during such displacement, i.e. that the distance
13 between the ends of the arms of each pair is constant during such radial
14 displacement and that no torsional forces are exerted on the arms during such
15 radial displacement.

16 In the currently preferred embodiment of the invention shown in Figs.
17 1-9, this is achieved as follows:

18 Each arm is provided with an angled end portion 45 having a guiding
19 pin 46 extending therethrough and projecting from both opposed surfaces of
20 the angled portion 45. The plane of each end portion 45 is substantially
21 parallel to the plane of the body 38 of the respective blade.

22 The rings of each pair of rings, for instance 15 and 16 in Fig. 6 or 9
23 and 10 in Fig. 8-9, are identical, and one lateral surface of each ring is
24 provided with a recessed circumferentially extending track 47 in the bottom of
25 an annular circumferentially extending recess 48 and an elongate radially
26 extending track 49 in the bottom of an annular circumferentially extending
27 recess 50 identical to the recess 48 and arranged diametrically opposite the
28 recess 48.

1 The two rings 15, 16 in Fig. 6 and the two rings 9, 10 in Figs 8 and 9
2 are arranged abutting each other with the lateral surfaces thereof provided
3 with the recesses 48 and 50 facing one another such that the recess 48 of the
4 ring 15 (ring 9) faces and overlies the recess 50 of the ring 16 (ring 10), and
5 the recess 50 of the ring 15 (ring 9) faces and overlies the recess 48 of the ring
6 16 (ring 10). Hereby annular channels 51 for receiving the angled end
7 portions 45 of the arms are formed when the rings of a pair 9/10, 11/12, 13/14
8 or 15/16 are arranged abutting each other.

9 One of the two projecting ends of each guiding pin 46 of each end
10 portion 45 is inserted in the circumferential track 47 of one ring of a pair of
11 rings while the other projecting end is inserted in the radial track 49 of the
12 other ring of said pair of rings.

13 The geometries of the tracks 47 and 49 are such that when one ring of a
14 pair of rings is rotated relative to the other ring of the pair, then the respective
15 body 38 of the blade carried by the pair of rings in question is displaced
16 radially such that the distance between the pins 46 of the two arms of the
17 respective blade remains constant and the arms are not subjected to any
18 torsional stresses.

19 In Figs. 8 and 9 the ring pair 9/10 is shown with the ring 9 abutting and
20 overlying the ring 10. In the illustration both rings are shown in full lines for
21 the sake of clarity and to illustrate the relative positions of the tracks 47 and
22 49 of both rings.

23 In Fig. 8 the ring 10 has been turned 10 degrees clockwise such that the
24 track 47 thereof shown at left in Fig. 8 is turned 10 degrees clockwise, while
25 the ring 9 has been turned 10 degrees counterclockwise so that the track 47
26 thereof shown at right in Fig. 8 is turned 10 degrees counterclockwise.
27 Consequently the track 49 of the ring 10 shown at right in Fig. 8 is turned 10
28 degrees clockwise while the track 49 of the ring 9 shown at left in Fig. 8 is
29 turned 10 degrees counterclockwise. The angles clockwise and

1 counterclockwise are given relative to an initial position where the body 38 is
2 at the halfway position between Fig.8 and Fig.9. The maximum periphery of
3 the light beam is shown by the circle 52.

4 In Fig. 9 the ring 10 has been turned 10 degrees counterclockwise such
5 that the track 47 thereof shown at left in Fig. 9 is turned 10 degrees
6 counterclockwise, while the ring 9 has been turned 10 degrees clockwise so
7 that the track 47 thereof shown at right in Fig. 9 is turned 10 degrees
8 clockwise. Consequently the track 49 of the ring 10 shown at right in Fig. 9 is
9 turned 10 degrees counterclockwise, while the track 49 of the ring 9 shown at
10 left in Fig. 9 is turned 10 degrees clockwise.

11 All intermediate positions between the two end positions shown in
12 Figs. 8 and 9 are achieved by rotating the rings 9 and 10 relative to one
13 another the corresponding amount of degrees between zero and twenty.

14 A multitude of different beam periphery shapes may be achieved by
15 displacing the blades 30-33 radially by rotating the two rings of the
16 corresponding ring pair relative to one another and by displacing the blades
17 circumferentially by rotating the two rings of a ring pair together.

18 In Fig. 7 one of infinitely many combinations of radial and
19 circumferential positions of the four blades 30-33 is shown, whereby a beam
20 44 with the shown eight sided polygonal peripheral shape is achieved.

21 So as to achieve a distance between the two pins 46 at the ends of the
22 two arms of each of the blades 30-33 that is the same for all radial
23 displacements of the body 38 thereof, and so as to provide that no torsion of
24 the arms takes place such that the body 38 is not subjected to any distorting
25 forces, the shapes of the tracks 47 and 49 are configured accordingly as
26 described in the following, with reference to Fig. 10 which illustrates the
27 construction and calculation of the said shapes of the tracks 47 and 49.

28 In Fig. 10 three pairs of mutually corresponding points on the curves
29 47 and 49 are constructed, the angles being exaggerated for the sake of clarity.

1 The construction of the curves is carried out according to the
2 following:

3 A1 is constant and equal to half the distance between the two pins 48
4 of a blade.

5 $C2 = A1$

6 $\text{Angle}1 = \text{Angle}2$

7 $\text{Angle}1 + \text{Angle}2 = \text{Angle}3$

8 Both triangles are right-angled triangles

9 Angle 1 is the angle at which ring 1 is set, and Angle 2 is the angle at
10 which ring 2 is set

11 By rotating ring 1 relative to ring 2, Angle 3 is obtained. A center line
12 is constructed from the center of the rings and horizontally to the left such that
13 $\text{Angle}1 = \text{Angle}2$.

14 Angle 1 and Angle 2 are used to construct two triangles.

15 A line is drawn along the center line, the line having a length equal to
16 half the length between the two pins 46 of a blade.

17 This line forms the hypotenuse C2 as well as the triangle side A1 so
18 that the other triangle side B1 can be constructed by drawing a line from the
19 right angle downwards and C1 away from the center until the two lines
20 intersect at a point. This point is on the curve to be constructed for configuring
21 track 47.

22 Equation 1.1: $B1 = \text{SIN}(\text{Angle}1) \times A1$

23 Equation 1.2: $C1 = A1/\text{COS}(\text{Angle}1)$

24

25 C1 is now a radius which together with Angle 3 may be used to construct
26 the track by means of the equations 1.3:

27 $X_{\text{track}47} = \text{COS}(\text{Angle}3) \times C1$

28 $Y_{\text{track}47} = \text{SIN}(\text{Angle}3) \times C1$

29 Or the equation 1.2 may be inserted in the equation 1.3:

1 position of the blade 60 is amended to the non-obstructing position thereof by
2 rotating the rings 56 and 57 relative to one another, for instance as shown by
3 rotating the ring 56 counterclockwise and maintaining the ring 57 in the same
4 position. Hereby the pin 62 will be forced to slide in the track 63 while the pin
5 61 merely rotates such that the blade rotates around the pin 61. In the shown
6 example a rotation of the ring 56 counterclockwise 12 degrees will result in a
7 rotation of 22 degrees of the blade 60.

8 This arrangement of the beam shaping blades requires relatively stiff
9 blades and/or relatively large axial spacing between the individual blades so
10 that the blades will not interfere with or engage one another when being
11 rotated.

12 Referring now to Figs. 2, 3, 12 and 13, the mechanism for displacing
13 the focusing lens 22 and the zoom lens 23 along the longitudinal axis 19 is
14 shown in partly exploded form. A holder 64 for the zoom lens 23 and a
15 holder 65 for the focusing lens 22 are slidably arranged in tracks 66 and 67,
16 respectively, in track rails so that the holders 64 and 65 may be displaced to
17 and fro parallel to the longitudinal axis 19.

18 A bracket 68 is connected to each of the holders 64 and 65, only the
19 bracket 68 for the holder 65 being visible. The brackets are each connected to
20 a respective toothed belt 69 and 70 corresponding to the holders 65 and 64,
21 respectively. The toothed belts are mounted on pulleys 71 and 72 rotatably
22 mounted on the track rails 66, 67.

23 Each of the adjustment rings 17 and 18 (partly cut away for clarity in
24 Fig. 12) are provided with lateral toothed portions 73 and 74, respectively, for
25 engaging the teeth of the toothed belts 69 and 70, respectively, so that rotation
26 of the ring 17 to and fro will cause displacement of the toothed belt 69 to and
27 fro, and rotation to and fro of the ring 18 will cause displacement to and fro of
28 the toothed belt 70. Hereby, the lens holders 64 and 65 may be displaced to
29 and fro along the tracks 66 and 67 by rotation to and fro of the rings 18 and

1 toothed belt and for displacement to and fro in the plane of said toothed belt
2 70, perpendicularly to said two parallel lengths. The locking sled is provided
3 with locking teeth 94a and 94b for locking engagement with teeth at the rim
4 of locking wheel 90 in a ratchet type action. If the locking sled 94 is in a
5 central position, i.e. not displaced toward any of the two parallel lengths of the
6 belt 70, then the locking teeth 94a and 94b will not engage the teeth of the
7 locking wheel 90 so no friction brake is applied to the belt 70.

8 The dimension of the locking sled 94 perpendicular to the parallel
9 lengths of the belt 70 is slightly longer than the distance between the common
10 tangents of the pulleys 71 and 72 such that in the central position of the
11 locking sled 94, the locking sled will press against the parallel lengths of the
12 belt 70.

13 If tension is applied to one of the parallel lengths of the toothed belt 70
14 because of the weight of the lens, said length will be tightened and the parallel
15 length will be loosened whereby the locking sled 94 will be displaced from
16 the central position to a lateral position where the respective one of the
17 locking teeth 94a and 94b will engage the ratchet teeth of the locking wheel
18 90, thereby applying frictional braking forces to the pulley 71 through the
19 friction washer 91.

20 However, if tension in one of the parallel lengths of the belt 70 is
21 caused by rotation of the ring 18 for axial displacement of the holder 64, then
22 the displacement of the locking sled 94 from the central position thereof will
23 not cause engagement of one of the locking teeth 94a or 94b with the ratchet
24 teeth of the locking wheel 90 as the ratchet effect will cause the respective
25 locking tooth to "ratchet" over the ratchet teeth.

26 Hereby, a selective braking mechanism is achieved whereby the brake
27 effect is operative, when the weight of the lens tries to rotate the respective
28 adjustment rings, but the brake effect is inoperative when rotation of the
29 respective ring is carried out to displace the lens axially.

1 It will be apparent to those skilled in the art that the principles of the
2 above selective braking mechanism may be applied in all applications where a
3 braking effect is required in one direction of force application and is not
4 required in the opposite direction of force application.

5 The arrangement of the gobo or pattern 29 in the ring 8 and the iris 28
6 in the ring 7 need not be described herein as it will be apparent to those
7 skilled in the art that this can be done in many ways well known in the art.

8 For remote control of the adjustment rings it will also be readily
9 apparent to those skilled in the art that an electrical motor with a pinion for
10 each ring may be arranged such that the teeth of the pinion mesh with the
11 teeth on the rim of the respective ring. The motors may for instance be firmly
12 attached to the frame 6 or be spring biased so that any irregularities in the
13 mounting of the rings and thereby the toothed rims may be taken up. Magnetic
14 markers may be attached to the rings such that a sensing means may sense the
15 marker and thereby precisely identify the position of the respective ring as a
16 basis for the subsequent rotation thereof to a new setting of the respective
17 beam influencing means.

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